

TO: H. S. Dewey, Process Engineer
 FROM: F. J. Newman, Power Plant Superintendent

In accordance with the instructions of Mr. L. P. McGillicudy, the following information on utilities is forwarded:

Steam

There are available 13,000 lb./hr. of 150 lb./sq. in. gauge steam @ 45¢/1,000 lb. and 1,500 lb./hr. of 15 lb./sq. in. gauge steam @ 15¢/1,000 lb. Steam requirements above those available will necessitate the erection of additional steam-producing facilities at the rate listed in Table 2. Steam which comes in direct contact with propane or tallow cannot be returned to the boiler and must be exhausted to the atmosphere or fed to the sewer. Steam lost in this manner will be taken as a loss of 10¢/1,000 lb.

Cooling Water

Cooling water is available at a pressure such that pumps will not be required for circulating the water through process equipment. The cost of cooling water is 3.0¢/1,000 gal. for any amount required. The water is available at 90°F. and cannot be returned above 105°F.

City water @ 8¢/1,000 gal. should be used in direct contact with propane or tallow and after being used must be fed to the sewer.

Electrical Power

Electrical power is available at the rate of 1¢/kw.-hr. in any quantity required.

TABLE 2

COST OF STEAM IF ADDITIONAL FACILITIES ARE REQUIRED

(These figures include investment and operating costs for the new installation.)

Steam, lb./hr.	Cost/1,000 lb. 15 lb./sq.in.gauge	Cost/1,000 lb. 150 lb./sq.in.gauge
1,000	\$0.78	\$1.05
5,000	0.68	0.90
10,000	0.58	0.78
15,000	0.52	0.70
20,000	0.48	0.65
25,000	0.42	0.60

TO: H. S. Dewey, Process Engineer
 FROM: W. L. Curtis, Accountant

At the request of Mr. L. P. McGillicudy I am sending you information regarding the economics of the proposed tallow-decolorization plant.

Fixed charges on per annum basis

Depreciation based on a 10-yr. pay-off will be 10% of constructed cost.

Miscellaneous taxes and insurance will be 1 1/2% of constructed cost.

Interest will be 4 1/2% of constructed cost.
 Overhead will be 6% of constructed cost.
 Labor, including pay-roll burden, has been estimated to be \$95,000/yr.
 Maintenance has been estimated to be \$35,000/yr.
 Revenues
 Color bodies are estimated to be worth 1¢/lb.

SOLUTION

Robert P. Bannon, University of Illinois

PLANT MEMORANDUM

TO: L. P. McGillicudy, Chief Engineer
 FROM: H. S. Dewey, Process Engineer
 RE: PROPANE EXTRACTION PLANT FOR DECOLORIZATION OF TALLOW

Decolorized tallow can be produced by the propane extraction process for 0.305 cent/lb. The present caustic process costs 0.499 cent/lb. At your request a propane extraction plant has been designed to decolorize 8,600 lb./hr. of animal tallow producing 8,170 lb./hr. of decolorized tallow (Lovibond color of three) and 430 lb./hr. of color bodies. The costs of the proposed plant are as follows:

Total constructed cost	\$139,110
Net annual cost	185,450
Annual fixed charges	30,600
Annual operating cost	186,850
Annual by-product revenue	32,000

A complete report on the design of the proposed plant is attached hereto.

The construction of this plant is strongly recommended.

INTRODUCTION

A liquid-liquid extraction process for the decolorization of animal tallow has been developed in the laboratory. The solvent for the new process is liquid propane under pressure. In order to determine whether the propane extraction method is more economical than the present caustic method (0.499 cent/lb.), a plant capable of treating 8,600 lb./hr. of raw tallow was designed and the cost estimated. The designed output is 8,170 lb./hr. of decolorized tallow (Lovibond color of three) and 430 lb./hr. of color bodies.

DESCRIPTION OF THE PROCESS

Raw tallow stored at 125°F. and atmospheric pressure (Figure 1) is pumped (P1) through exchanger E9, in which it is heated to 165°F., to the top baffle of a continuous countercurrent propane extraction tower (T1). The extraction tower is operated isothermally at 165°F. and 450 lb./sq.in.abs. tower-top pressure. Extract consist-

ing of propane and refined tallow is removed as overhead, and raffinate composed of color bodies and propane comprise the bottoms.

Overhead Stream. The overhead stream is heated (E1) to 175°F. with 150-lb. steam. Slightly less than half the solvent in the overhead is vaporized here. This vapor is separated from the liquid in flash drum D1 (450 lb./sq.in.abs.) and is condensed in E2 with tallow and boiling propane from D1 or in E3 with cooling water. From the condenser the propane flows back to the storage tank. The liquid from D1 is throttled to 350 lb./sq.in.abs. and passed through exchanger E2, where most of the remaining solvent is vaporized. This mixture of liquid and vapor is separated in drum D2 (350 lb./sq.in.abs. and 152°F.). Vapor from D2 is condensed with cooling water in E5 at 350 lb./sq.in., flows through holding tank H2, and is pumped (P3) back to the propane storage tank. The liquid from drum D2 is throttled to 230 lb./sq.in.abs. and heated (E4) to 252°F. with 150-lb. steam. The vapor formed in E4 is separated from the tallow in flash drum D3. The vapor from D3 is condensed with cooling water (E6), flows through holding tank H1, and is pumped back to the propane storage tank. The refined-tallow stream from D3 is throttled down to 17 lb./sq.in.abs. and into steam stripper S1 where 150-lb. steam strips out the remaining solvent. The tallow from S1 flows by gravity and stripper pressure through exchanger E9, where it heats up raw tallow feed; through exchanger E10, where it is further cooled to its storage temperature with cooling water; and then to the refined-tallow storage tank. The mixture of steam and propane vapor from stripper S1 is separated in a jet condenser J1. The propane vapor from J1 is compressed (C1) to 230 lb./sq.in.abs.,

condensed (E6) with cooling water, and pumped (P4) to the storage tank.

Bottoms Stream. The bottoms stream of propane and color bodies from the extraction tower (T1) is throttled to 230 lb./sq.in.abs. and heated (E8) with 150-lb. steam to 237°F. The vapor formed is separated in flash drum D4. This vapor is also condensed in E6 and pumped (P4) back to the propane storage tank. The liquid from D4 is throttled down to 17 lb./sq.in.abs. and into steam stripper S2, where 150-lb. steam strips out the remaining solvent. The color bodies from S2 flow by gravity and stripper pressure through a cooler (E7) to storage. The mixture of steam and vapor from stripper S2 joins the stream from stripper S1 and goes through the jet condenser as described.

EQUIPMENT LIST

I. Extraction Tower T1

Diameter: 8.5 ft.

Height: 50.5 ft.

No. of baffles: 26

Operating temperature and pressure:
165°F. and 450 lb./sq.in.abs.

Cost: \$30,704.

II. Flash Drums

	D1	D2	D3	D4
Diameter, ft.	4.5	4.0	1.5	0.5
Height, ft.	21.0	9.0	11.5	6.0
Operating temp., °F.	175	152	252	237
Operating press., lb./sq.in.abs.	450	350	230	230
Cost, \$	5,623	1,580	697	128

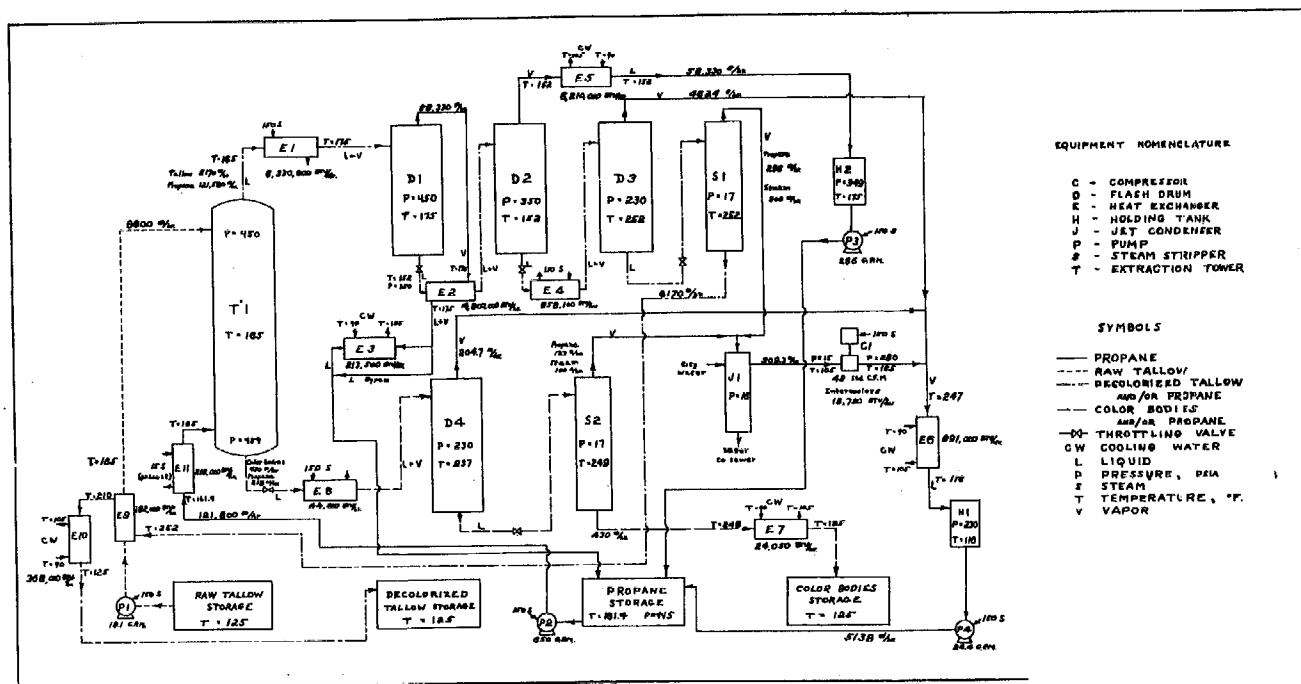


Fig. 1. Flow sheet for proposed tallow-decolorization plant.

III. Steam Strippers -- Packed with 1-in. Raschig Rings

	S1	S2
Diameter, ft.	1.0	1.0
Height, ft.	25	25
Operating temp., °F.	252	249
Operating press., lb./sq.in.abs.	17	17
Cost, \$	397.50	397.50

IV. Jet Condenser J1

Diameter: 0.5 ft.
 Height: 17 ft.
 Operating pressure: 16 lb./sq.in.abs.
 Cost: \$222

VIII. Holding Tanks

	H1	H2
Height, ft.	6.0	9.5
Diameter, ft.	2.0	5.0
Operating pressure, lb./sq.in.abs.	230	349
Cost, \$	370	1,900

IX. Propane Storage Tank

Length: 25 ft.
 Diameter: 9 ft.
 Operating pressure: 445 lb./sq.in.abs.
 Cost: Already paid out

Judging Committee comment: This item cannot be neglected; only tallow storage has been paid out.

V. Heat Exchangers: Shell and Tube

Exchanger duty	B.t.u./hr.	ΔT, °F.	U, B.t.u./(hr.)(ft./°F.)	Area, ft.	Max. operating pressure, lb./sq.in.abs.	Cost, \$
E1	6,330,000	Heating 196 Vapor. 191	202 204	162	450	2,485
E2	4,800,000	23	91.2	2,290	450	11,700
E3	213,500	77.5	171.5	16	450	350
E4	858,100	158.5	220	16	230	256
E5	6,210,000	54.5	171.5	665	350	3,840
E6	991,000	desup. 68 cond. 17.5	14.3 171.5	563	230	3,740
E7	24,050	73	42.9	7.7	15	109
E8	44,500	165	217	1.24	230	28
E9	182,000	86	25	85	450	1,820
E10	368,000	63.6	42.9	135	15	1,335
E11	219,000	87.5	107	23.4	450	512

VI. Compressor C1 - Steam-driven Reciprocating Type, Three Stages with Intercoolers

Capacity: 42.0 std.cu.ft./min.
 Power delivered: 10.7 hp.
 Operating range: 15 to 230 lb./sq.in.abs.
 Cost: \$1,070

COST SUMMARY

Total constructed cost	\$139,110
Annual investment cost	\$ 30,600
Annual operating costs:	
Utilities	\$53,000
Propane	3,850
Labor	95,000
Maintenance	35,000
Total	\$186,850
Total annual cost	\$217,450
Annual revenue:	
Value of color bodies	\$ 32,000
Net annual cost	\$185,450
Net cost per pound of tallow produced	\$0.00305
Net annual saving of extraction process over caustic process	\$118,050

VII. Pumps - Steam-driven Reciprocating Type

	P1	P2	P3	P4
Capacity, gal./min.	19.1	650	286	24.4
Pressure head, lb./sq.in.abs.	450	19	100	115
Power delivered, hp.	5.05	5.09	16.2	2.81
Cost, \$	580	585	1,555	534

DISCUSSION OF THE PROCESS

The process design must necessarily start with the extraction tower. From the equilibrium curves it is seen that, as the temperature increases, the concentration differences between phases in equilibrium increases, and therefore fewer equilibrium stages would be needed for a given separation at 165° than at 160°F. or lower. For this reason the tower was designed for isothermal operation at 165°F. The tower-top temperature was necessarily fixed at 165°F. owing to miscibility considerations.

In a solvent-recovery system a large quantity of heat is necessary to vaporize the propane. In order to make operating costs as low as possible, it is necessary to recover as much heat from the propane as possible. The propane-recovery process is essentially an evaporation process with several complications. According to the equilibrium constant nomograph in reference 2, the lower the pressure, the higher is the temperature necessary for a given liquid-vapor equilibrium. This would seem to indicate that the multiple-effect evaporator principle could not be used; however, these equilibrium constants do not hold for solutions containing a substance of high and one of low molecular weight when the mole fraction of the high-molecular-weight substance is negligible (5). It is possible, therefore, to realize the heat economy of multiple-effect evaporation in the first two flash drums of the overhead propane-recovery system.

Slightly less than one half the propane in the overhead stream was vaporized at 450 lb./sq.in. abs. and 175°F. with 150 lb. steam, and this vapor was used as a heat source to vaporize most of the remaining solvent at 350 lb./sq.in.abs. and 152°F. (One-hundred-and-fifty-pound steam was used in E1 because it was cheaper than 15-pound steam for the quantity needed.) A little more propane was vaporized in E1 than was necessary to use in E2. This was done to permit better and much easier control of the process. It is a little more expensive to do this, but it greatly reduces the difficulty of control inherent in this type of design. An additional exchanger (E3) was provided to condense this excess vapor. The pressure for the second flash drum (D2) was set at 350 lb./sq.in. abs., which was considered to be the best pressure for a balance between a ΔT on the exchanger (E2) preceding the drum and the amount of propane that could be separated in the drum. Since the mole fraction of tallow in the liquid in drum D2 is appreciable, an equilibrium constant was used to determine the propane concentration in the liquid. A third flash drum was necessary to reduce the solvent concentration to the required value for entering stripper S1. This third drum was operated at as low a pressure as possible that would permit condensation of the vapor with cooling water. This low pressure resulted in the temperature in the drum being as low as possible, and steam was thereby saved. The bottoms stream was small and contained so little solvent that only one flash drum at 230 lb./sq.in.abs. was needed.

Holding or surge tanks were considered to be necessary between propane condensers and pumps. Without tanks at these locations surging would tend to decrease the effective area of the condenser because of flooding.

The vapor streams from the two strippers were combined so that only one jet condenser was needed. Practically all water introduced into the propane stream in the jet condenser will be removed in the intercoolers of compressor C1. However, over a considerable time there may be a build-up of water in the propane system. Since propane and water are slightly miscible, this water could not be separated out in holding tank H1.

The color bodies leaving stripper S2 were cooled to 125°F. before being sent to storage. This may or may not be necessary depending on subsequent processes and the holding time in storage.

APPENDIX

Discussion of Calculations

Only the calculations for the final design are included in this report. Several other designs, from which the final design evolved, were partially calculated.

Each piece of equipment is calculated separately and assumptions made are stated for the first of each type of equipment. General assumptions made are

1. Heat losses are negligible.
2. No tallow or color bodies are vaporized in any equipment or lost in any other way.

Thermodynamic data for propane were obtained from references 1 and 3 and vapor-liquid equilibrium constants from reference 2. Thermodynamic data for steam were obtained from steam tables.

DESIGN CALCULATIONS

I. Extraction tower

Material balances (S.F.B.)

$$\begin{aligned} \text{Over-all balance: } \text{Feed} &= \text{raffinate} + \text{extract} \\ 8,600 &= R + 8,170 \\ R &= 430 \text{ lb./hr.} \end{aligned}$$

$$\begin{aligned} \text{Tallow balance } \quad FX_F &= RX_R + EY_E \\ (8,600)(0.860) &= 430X_R + (8,170 \times 0.894) \\ X_R &= 0.214 \end{aligned}$$

$$\text{Mole balance } \quad N_F = N_R + N_E$$

$$\frac{8,600}{750} = N_R + \frac{8,170}{740}$$

$$N_R = 0.4265 \text{ lb. mole/hr.}$$

$$\text{M.W. bottoms} = \frac{430}{0.4265} = 1,008$$

From Ponchon - Savarit diagram (Figure 2)

$$\frac{S_P}{R} = 283$$

$$S_P = 283 \cdot 430 = 121,800 \text{ lb./hr.}$$

$$\begin{aligned} \text{Solvent out in bottoms} &= \\ 430 \times 0.5057 &= 218 \text{ lb./hr.} \end{aligned}$$

$$\begin{aligned} \text{Solvent out in overhead} &= \\ 121,800 - 218 &= 121,580 \text{ lb./hr.} \end{aligned}$$

$$\text{Number of theoretical stages} = 3.88$$

$$\text{Number of baffles} =$$

$$\frac{\text{No. of theoretical stages}}{\text{baffle efficiency}} = \frac{3.88}{0.15} = 25.9$$

∴ 26 baffles are needed.

Since tower is isothermal, no heating coils are necessary and the feed tray is the top tray.

$$\begin{aligned} \text{Height} &= (\text{No. of baffles} - 1) 1.5 \\ &+ \text{height of settling sections} \\ &= (26 - 1) 1.5 + 8 + 5 = 50.5 \text{ ft.} \end{aligned}$$

Since 450 lb./sq.in.abs. is the minimum allowable pressure, this will be the tower top pressure.

$$\text{Tower bottom pressure} = 450 + \text{liquid head}$$

$$= 450 + \text{height} \times \text{average density} \times \frac{1}{144}$$

$$\text{Average density} =$$

$$\frac{8,600 \cdot 0.906 - 3 \times 10^{-4} (165 - 100) 62.4 + 121,800 \times 24.4}{8,600 + 121,800}$$

$$= 26.4 \text{ lb./cu.ft.}$$

$$\text{Tower bottom pressure}$$

$$= 450 + 50.5 \times 26.4 \times \frac{1}{144} = 459 \text{ lb./sq.in.abs.}$$

Diameter

$$\begin{aligned} \text{Volume of propane phase at top} &= \\ &= \text{volume of propane} + \text{volume of extract} \\ &= 121,580 \times 0.041 \end{aligned}$$

$$= \frac{8,170}{62.4 [0.905 - 3 \times 10^{-4} (165 - 100)]}$$

$$= 5,138 \text{ cu.ft./hr.}$$

$$\begin{aligned} \text{Volume of propane phase at propane} &= \\ \text{entrance} &= 121,800 \times 0.041 = 5,000 \text{ cu. ft. /hr.} \end{aligned}$$

Therefore, maximum superficial velocity will be at top of tower.

$$\begin{aligned} \text{Maximum allowable superficial velocity} &= \\ &= 90 \text{ ft./hr.} \end{aligned}$$

$$\text{Tower diameter} = 2 \sqrt{\frac{5,138}{90 \pi}} = 8.5 \text{ ft.}$$

Cost: It is assumed that interpolation of cost data is permissible for in-between diameters.

$$\text{Vessel cost} = 50.5 \times 152 \times 4.00 = \$30,704$$

$$\text{Baffle cost} = 26 \times 180 = \$ 4,680$$

$$\text{Total tower cost} = \$35,384$$

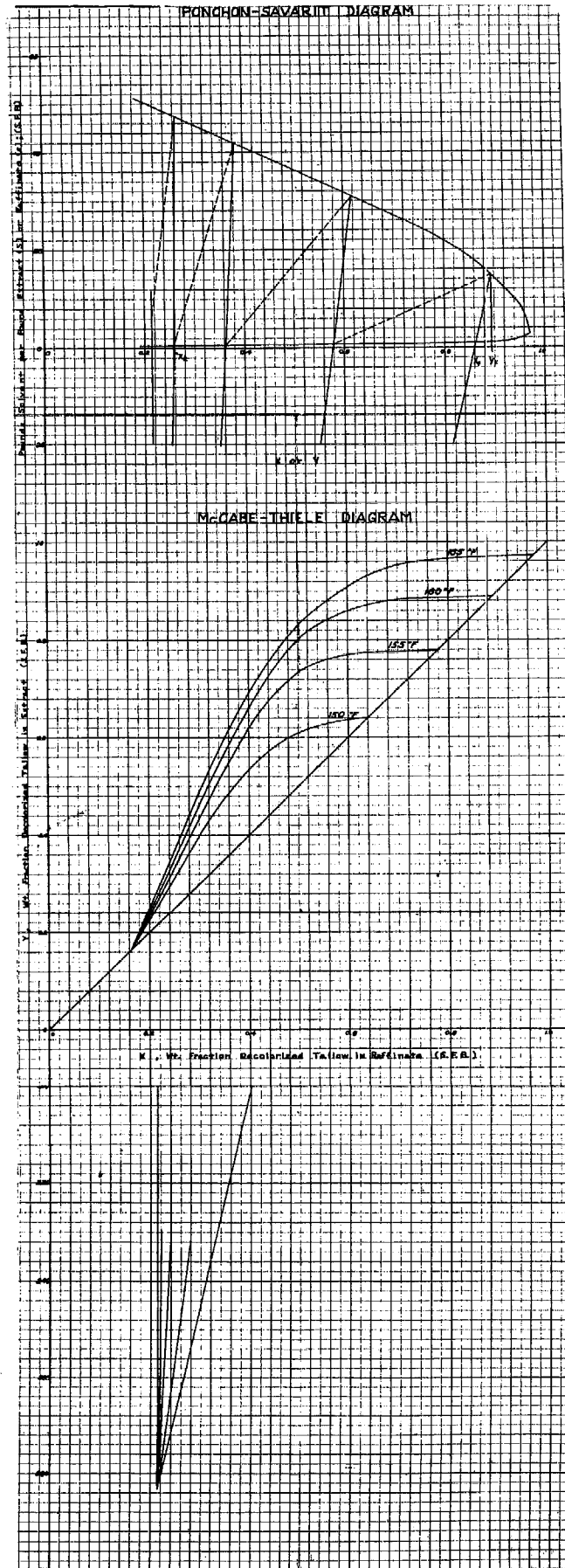


Fig. 2.

II. Overhead Flash Drums

Overhead stream

$$\text{Tallow} = 8,170 \text{ lb./hr.} = 11.04 \text{ lb. moles/hr.}$$

$$\text{Propane} = 121,580 \text{ lb./hr.} = 2,760 \text{ lb. moles/hr.}$$

It is desired to balance the amount of propane vaporized in exchanger E1 and E2 so that the vapor from flash drum D1 will be a little more than just sufficient to meet the heat requirements of E2.

Flash drum D2

$$\text{Pressure} = 350 \text{ lb./sq.in.abs.}$$

$$\text{Temperature} = 152^\circ\text{F.}$$

$K = 1.10$ from equilibrium constant nomograph (2); as the mole fraction of tallow is appreciable this K is assumed to hold.

$$\text{Mole fraction propane in vapor} = 1$$

$$\text{Mole fraction propane in liquid} = \frac{1}{K} = 0.91$$

$$\begin{aligned} \text{Moles in exit liquid stream} &= \frac{11.04}{1.0 - 0.91} \\ &= 123 \text{ lb. moles/hr.} \end{aligned}$$

Liquid propane leaving D2

$$= (123 - 11.04)44 = 4,920 \text{ lb./hr.}$$

Propane vapor separated in D1 and D2

$$= 121,580 - 4,920 = 116,660 \text{ lb./hr.}$$

$$x = \text{propane vaporized in E1, lb./hr.}$$

$$y = \text{propane vaporized in E2, lb./hr.}$$

Exchanger E2 duty

$$\begin{aligned} &= y(408.4 - 322.5) + 4,920(301.8 - 322.5) \\ &\quad + 8,170 \times 0.53 (152 - 175) \\ &= 85.9y - 201,600 \text{ B.t.u./hr.} \end{aligned}$$

Heat available from propane vapor from D1

$$= x(408.5 - 322.5) = 86.0x$$

$$\text{Therefore: } 86.0x = 85.9y - 201,600$$

$$x + y = 116,660$$

$$x = 57,160 \text{ lb./hr.}$$

$$y = 59,500 \text{ lb./hr.}$$

In order to provide an extra margin of heat in exchanger E2 for better control of the process, x and y will be made equal.

$$x = y = \frac{1}{2} \cdot 116,660 = 58,330 \text{ lb./hr.}$$

1. Flash drum D1

$$\text{Pressure} = 450 \text{ lb./sq.in.abs.}$$

Temperature = 175°F. , the boiling point of pure propane at 450 lb./sq.in.abs.

It is assumed that the presence of the tallow does not affect this temperature appreciably. Equilibrium constants do not hold when the mole fraction of the tallow is negligible, as is the case here. The solution is treated as if it were pure propane.

Exit streams:

$$\text{Propane vapor} = x = 58,330 \text{ lb./hr.}$$

$$\begin{aligned} \text{Propane liquid} &= 121,580 - 58,330 \\ &= 63,250 \text{ lb./hr.} \end{aligned}$$

$$\text{Tallow} = 8,170 \text{ lb./hr.}$$

Diameter:

$$\text{Vapor rate} = 58,330 \text{ lb./hr.} = 16.2 \text{ lb./sec.}$$

$$\text{Sp. vol. of vapor} = 0.194 \text{ cu.ft./lb.}$$

$$= 16.2 \cdot 0.194 = 3.14 \text{ cu.ft./sec.}$$

$$\rho_G = \frac{1}{\text{sp.vol.}} = 5.15 \text{ lb./cu.ft.}$$

$$\begin{aligned} \rho_{\text{tallow}} &= [0.905 - 3 \times 10^{-4}(175 - 100)] 62.4 \\ &= 55.0 \text{ lb./cu.ft.} \end{aligned}$$

$$\rho_{\text{propane liquid}} = \frac{1}{0.0425} = 23.5 \text{ lb./cu.ft.}$$

$$\begin{aligned} \rho_{L_{\text{avg.}}} &= \frac{55.0 \times 8,170 + 23.5 \times 63,250}{8,170 + 63,250} \\ &= 27.1 \text{ lb./cu.ft.} \end{aligned}$$

Judging Committee comment:
Specific volumes are additive
not densities.

$$D^2 = \frac{8 \times \text{cu.ft./sec.}}{0.227 \pi \frac{\rho_L - \rho_G}{\rho_G}} = \frac{8 \times 3.14}{0.227 \pi \frac{27.1 - 5.15}{5.15}}$$

$$D = 4.13 \text{ ft. to nearest } \frac{1}{2} \text{ ft.} = 4.5 \text{ ft.}$$

Height:

$$h_L = \frac{(8,170 + 63,250)1/12}{27.1 \times \pi \left(\frac{4.5}{2}\right)^2} = 13.8 \text{ ft.}$$

$$\begin{aligned} h &= 2 + D \times 1.17 + h_L = 2 + 4.5 \times 1.17 + 13.8 \\ &= 21.0 \text{ ft.} \end{aligned}$$

Cost:

$$\text{Vessel cost} = 21 \times 64 \times 4.00 = \$5,375$$

$$\text{Mist separator cost} = 4 \times 62 = 248$$

$$\text{Total cost} = \$5,623$$

2. Flash drum D2

Diameter:

$$\text{Vapor rate} = 58,330 \text{ lb./hr.} = 16.2 \text{ lb./sec.}$$

$$\text{Sp. vol. of vapor} = 0.270 \text{ cu.ft./lb.}$$

$$= 0.270 \times 16.2 = 4.38 \text{ cu.ft./sec.}$$

$$\rho_G = \frac{1}{\text{sp.vol.}} = 3.70 \text{ lb./cu.ft.}$$

$$\begin{aligned} \rho_{\text{tallow}} &= [0.905 - 3 \times 10^{-4}(152 - 100)] 62.4 \\ &= 55.4 \text{ lb./cu.ft.} \end{aligned}$$

$$\rho_{\text{propane liquid}} = 26.0 \text{ lb./cu.ft.}$$

$$\begin{aligned} \rho_{L_{\text{avg.}}} &= \frac{55.4 \times 8,170 + 26.0 \times 4,920}{8,170 + 4,920} \\ &= 44.3 \text{ lb./cu.ft.} \end{aligned}$$

Judging Committee comment: Specific
volumes are additive not densities.

$$D^2 = \frac{8 \cdot 4.38}{0.227 \pi \sqrt{\frac{44.3 - 3.70}{3.70}}}$$

D = 3.85 ft. to nearest $\frac{1}{2}$ ft. = 4.0 ft.

Height:

$$h_L = \frac{8,170 + 4,920}{44.3 \cdot \pi \left(\frac{4}{2}\right)^2} = 1.96 \text{ ft.}$$

$$h = 2 + 4.0 \cdot 1.17 + 1.96 = 9.0 \text{ ft.}$$

Cost:

$$\text{Vessel cost} = 9.0 \times 56 \times 2.70 = \$1,360$$

$$\text{Mist separator cost} = 4 \times 55 = \$ 220$$

$$\text{Total cost} = \$1,580$$

3. Flash drum D3

Pressure = 230 lb./sq.in.abs.

Entering stream:

$$\text{Tallow} = 8,170 \text{ lb./hr.} = 11.04 \text{ lb. moles/hr.}$$

$$\text{Propane} = 4,920 \text{ lb./hr.} = 112 \text{ lb. moles/hr.}$$

Exit streams:

$$\text{Tallow} = 8,170 \text{ lb./hr.}$$

$$\begin{aligned} \text{Propane liquid} &= 0.035 \times \frac{8,170}{1.0 - 0.035} \\ &= 296 \text{ lb./hr.} \end{aligned}$$

$$\text{Propane vapor} = 4,920 - 296 = 4,624 \text{ lb./hr.}$$

Mole fraction propane in exit liquid

$$\begin{aligned} &= \frac{296}{44} = 0.379 \\ &= \frac{296}{44 + 11.04} \end{aligned}$$

$$K = \frac{y^1}{x^1} = \frac{1}{0.379} = 2.64$$

∴ Temperature = 252°F. [equil. constant nomograph (2)]

Diameter:

$$\text{Vapor rate} = 4,624 \text{ lb./hr.} = 1.285 \text{ lb./sec.}$$

$$\text{Sp. vol. vapor} = 0.62 \text{ cu.ft./lb.}$$

$$= 0.62 \times 1.285 = 0.796 \text{ cu.ft./sec.}$$

$$\rho_G = \frac{1}{\text{sp.vol.}} = 1.61 \text{ lb./cu.ft.}$$

$$\begin{aligned} \rho_L &= [0.905 - 3 \times 10^{-4}(252 - 100)] 62.4 \\ &= 53.6 \text{ lb./cu.ft.} \end{aligned}$$

(Above propane critical temperature, density of solution is same as density of tallow.)

$$D^2 = \frac{8 \cdot 0.796}{0.227 \pi \sqrt{\frac{53.6 - 1.61}{1.61}}}$$

$$D = 1.25 \text{ ft. to nearest } \frac{1}{2} \text{ ft.} = 1.5 \text{ ft.}$$

Height:

$$h_L = \frac{(8,170 + 296)1/12}{53.6 \times \pi \left(\frac{1.5}{2}\right)^2} = 7.45 \text{ ft.}$$

$$h = 2 + (1.5 \times 1.17) + 7.45 = 11.2 \text{ ft.}$$

$$\text{to nearest } \frac{1}{2} \text{ ft.} = 11.5 \text{ ft.}$$

Cost:

$$\text{Vessel cost} = 11.5 \times 22 \times 2.20 = \$557$$

$$\text{Mist separator} = 4 \times 35 = \$140$$

$$\text{Total cost} = \$697$$

III. Overhead Steam Stripper S1

Pressure = 17 lb./sq.in.

Entering stream at 252°F.

Tallow = 8,170 lb./hr.

Propane = 296 lb./hr.

It is assumed that the temperature change on throttling from the flash-drum pressure to the stripper pressure is negligible because of the small percentage of propane in the stream.

Heat balance:

The propane is above its critical temperature (206°F.) and therefore has no heat of vaporization.

Approximate heat capacity for propane in this range is 0.5 B.t.u./(lb.)(°F.)

Heat necessary to raise temperature of tallow stream 1°F. = $8,170 \times 0.53 + 296 \times 0.5 = 4,477$ B.t.u./hr.

Heat given up by steam in cooling to incoming tallow stream temperature

$$= 300(1,195.6 - 1,169.3) = 7,900 \text{ B.t.u./hr.}$$

The steam could not raise the temperature of the tallow as much as two degrees. Therefore it is assumed that the tallow stream leaves at the same temperature it entered and that the heat given off by the steam is dissipated in heat effects owing to stripping and in losses.

Exit streams at 252°F. and 16 lb./sq.in.abs.

Liquid: tallow = 8,170 lb./hr.

negligible amount of propane

Vapor: steam = 300 lb./hr.

propane = 296 lb./hr.

Size (given): 25 ft. high and 1 ft. in diameter packed with 15 ft. of 1-in. Raschig rings

Cost:

$$\text{Vessel cost} = 25 \times 15 \times 0.90 = \$337.50$$

$$\text{Volume of packing} = \pi \left(\frac{1}{2}\right)^2 \times 15$$

$$= 11.8 \text{ cu.ft.}$$

round off to 12 cu.ft.

$$\text{Cost of packing} = 12 \times 5 = \$ 60.00$$

$$\text{Total stripper cost} = \$397.50$$

IV. Bottoms flash drum D4

Pressure = 230 lb./sq.in.abs.

Entering stream:

Color bodies = 430 lb./hr.

Propane = 218 lb./hr.

Exit streams:

Color bodies = 430 lb./hr.

$$\text{Propane liquid} = 0.03 \times \frac{430}{1.0 - 0.03}$$

$$= 13.3 \text{ lb./hr.}$$

Propane vapor = 218 - 13.3 = 204.7 lb./hr.

Mole fraction propane in exit liquid

$$= \frac{13.3}{\frac{13.3}{44} + \frac{430}{1,008}} = 0.415$$

$$K = \frac{y^1}{x^1} = \frac{1}{0.415} = 2.41$$

Temperature = 237°F. [equilibrium constant nomograph (2)]

Diameter:

Vapor rate = 204.7 lb./hr. = 0.057 lb./sec.

Sp. vol. of vapor = 0.62 cu.ft./lb.

$$= 0.62 \times 0.057 = 0.0354 \text{ cu.ft./sec.}$$

$$\rho_G = \frac{1}{\text{sp.vol.}} = 1.61 \text{ lb./cu.ft.}$$

$$\rho_L = [0.980 - 3 \times 10^{-4}(237 - 100)] 62.4 = 58.4 \text{ lb./cu.ft.}$$

$$D^2 = \frac{8 \times 0.0354}{0.227 \pi \sqrt{\frac{58.4 - 1.61}{1.61}}}$$

$$D = 0.26 \text{ ft. to nearest } \frac{1}{2} \text{ ft.} = 0.5 \text{ ft.}$$

Height:

$$h_L = \frac{(430 + 13.3)1/12}{58.4 \pi \left(\frac{0.5}{2}\right)^2} = 3.22 \text{ ft.}$$

$$h = 2 + 0.5 \cdot 1.17 + 3.22 = 6 \text{ ft.}$$

Cost:

Vessel cost = 6 × 8 × 2.20 = \$106

Mist separator = 4 × 20 = \$ 80

Total cost = \$186

V. Bottoms Steam Stripper S2

Pressure = 17 lb./sq.in.abs.

Entering stream at 237°F.

Color bodies = 430 lb./hr.

Propane = 13.3 lb./hr.

Heat balance: Assumptions made for overhead stripper cannot be made here because the bottoms stream is much smaller than the overhead stream.

Heat given up by steam = heat absorbed by bottoms stream

100 (ΔH steam) = 430 × 0.50 (t-237)

+ 13.3 × 0.5 (t-237)

By trial and error, t = 249°F.

Exit streams at 16 lb./sq.in.abs. and 249°F.

Liquid: color bodies = 430 lb./hr.

negligible amount of propane

Vapor: steam = 100 lb./hr.

propane = 13.3 lb./hr.

Size and cost same as for overhead stripper.

VI. Jet Condenser J1

Vapor streams from both strippers are joined and fed to a common jet condenser.

Balance to determine temperature of entering vapor:

300 (ΔH steam from t to 252°F.)

+ 296 × 0.5 (252-t)

- 100 (ΔH steam from 249 to t)

- 13.3 × 0.5 (t-249) = 0

t = 251°F.

Duty = 400(1,168.1 - 184.4) + 400(216 - 120)

+ 309.3 × 0.5(251-105) = 453,700 B.t.u./hr.

$$\text{City water consumption} = \frac{457,700}{1(120-90)8.33}$$

$$= 1,820 \text{ gal./hr.}$$

Diameter:

Maximum vapor velocity will occur at vapor entrance.

Sp. vol. steam = 26.34 cu.ft./lb.

$$\text{Sp. vol. propane} = \frac{NRT}{P} = \frac{1}{44} \times 10.73 \times 709$$

$$= 10.8 \text{ cu.ft./lb.}$$

$$\text{Average sp. vol.} = \frac{26.31 \times 400 + 10.8 \times 309.3}{400 + 309.3}$$

$$= 19.6 \text{ cu.ft./lb.}$$

$$\rho_G = \frac{1}{19.6} = 0.051 \text{ lb./cu.ft.}$$

$$\rho_L = 62.4 \text{ lb./cu.ft.}$$

$$\text{Max. vapor velocity} = \frac{0.136 \sqrt{\rho_L - \rho_G}}{\rho_G}$$

$$= \frac{0.136 \sqrt{62.4 - 0.051}}{0.051} = 21.1 \text{ ft./sec.}$$

$$\text{Cross-sectional area} = \frac{19.6 \times \frac{709.3}{3,600}}{21.1}$$

$$= 0.183 \text{ sq.ft.}$$

$$\text{Diameter} = 2 \sqrt{\frac{0.183}{\pi}} = 0.5 \text{ ft.}$$

Height = 17 ft. (given)

Cost = 17 × 8 × 0.90 + 5 × 20 = \$222

VII. Heat Exchangers

Shell-and-tube countercurrent flow. The thermal resistance of the tubes and the inside-outside area factor are neglected in all the exchangers.

E1:

Hot side: 150-lb. steam
 $t = 366^{\circ}\text{F}$.

Cold side: overhead stream from extraction tower

$t_{\text{in}} = 165^{\circ}\text{F}$.
 $t_{\text{out}} = 175^{\circ}\text{F}$.

For the quantity needed, 150-lb. steam was cheaper than 15-lb. steam.

Heating duty = $8,170 \times 0.53(175-165)$
 $+ 121,580(322.5-312) = 1,320,000 \text{ B.t.u./hr.}$

Vaporizing duty = $58,330(408.5-322.5)$
 $= 5,010,000 \text{ B.t.u./hr.}$

Total duty = $6,320,000 \text{ B.t.u./hr.}$

Steam consumption = $\frac{6,320,000}{857} = 7,370 \text{ lb./hr.}$

Exchanger must be divided into two parts to calculate area.

Heating:

In mean $\Delta T = 196^{\circ}\text{F}$.

$h_{\text{tallow+propane}} = 252 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$
 [Figure 3]

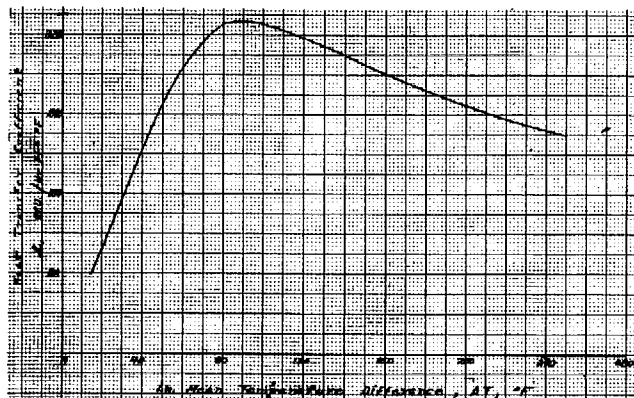


Fig. 3. Effect of ΔT on heat transfer coefficient for tallow plus boiling propane.

$$U = \frac{1}{\frac{1}{1,000} + \frac{1}{252}} = 202 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$$

$$\text{Area} = \frac{1,320,000}{202 \times 196} = 33.3 \text{ sq.ft.}$$

Vaporizing at 175°F .

$\Delta T = 191^{\circ}\text{F}$.

$h_{\text{tallow+propane}} = 256 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$

$$U = \frac{1}{\frac{1}{1,000} + \frac{1}{256}} = 204 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$$

$$\text{Area} = \frac{5,010,000}{204 \times 191} = 129 \text{ sq.ft.}$$

Total area = 162 sq.ft.

Cost = $162 \times 10.50 \times 1.46 = \$2,485$

It is assumed that cost data for exchangers cannot be interpolated.

E2:

Hot side: propane vapor from flash drum D1 $t = 175^{\circ}\text{F}$.

Cold side: tallow and liquid propane from flash drum D1 $t = 152^{\circ}\text{F}$.

Duty = $85.9 \times 58,330 - 201,600$
 $= 4,800,000 \text{ B.t.u./hr.}$

Vapor condensed = $\frac{4,800,000}{86.0} = 55,850 \text{ lb./hr.}$

It is assumed that on throttling before the stream goes through the exchanger the temperature drops to 152°F , the boiling point of propane at $350 \text{ lb./sq.in.abs}$.

$\Delta T = 175 - 152 = 23^{\circ}\text{F}$.

$$U = \frac{1}{\frac{1}{400} + \frac{1}{118}} = 91.2 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$$

$$\text{Area} = \frac{4,800,000}{91.2 \times 23} = 2,290 \text{ sq.ft.}$$

Cost = $2,290 \times 3.50 \times 1.46 = \$11,700$

E3:

This exchanger is used to condense any vapor not condensed in E2. The system is designed to give a slight excess of vapor over what is needed for heating in E2 in order to permit better control of the process.

Hot side: propane vapor from D1
 $t = 175^{\circ}\text{F}$.

Cold side: cooling water
 $t_{\text{in}} = 90^{\circ}\text{F}$.
 $t_{\text{out}} = 105^{\circ}\text{F}$.

Amount of propane condensed
 $= 58,330 - 55,850 = 2,480 \text{ lb./hr.}$
 Duty = $2,480(86.0) = 213,500 \text{ B.t.u./hr.}$

Cooling-water consumption = $\frac{213,500}{15 \times 8.33}$
 $= 1,710 \text{ gal./hr.}$

$\text{lm } \Delta T = 77.5^{\circ}\text{F}$.

$$U = \frac{1}{\frac{1}{300} + \frac{1}{400}} = 171.5 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F})$$

$$\text{Area} = \frac{213,500}{77.5 \times 171.5} = 16.0 \text{ sq.ft.}$$

Cost = $16.0 \times 15.00 \times 1.46 = \350

E4:

Hot side: 150-lb. steam

$$t = 366^{\circ}\text{F.}$$

Cold side: tallow and boiling propane
from drum D2

$$t_{\text{in}} = 152^{\circ}\text{F.}$$

$$t_{\text{out}} = 252^{\circ}\text{F.}$$

$$\begin{aligned} \text{Duty} &= (4,920 - 296)(475 - 302) + 296 \\ &\times 0.5(252 - 152) + 8,170 \times 0.53(252 - 152) \\ &= 858,100 \text{ B.t.u./hr.} \end{aligned}$$

$$\text{Steam consumption} = \frac{858,100}{.857} = 1,000 \text{ lb./hr.}$$

It is assumed that the boiling point of the
tallow-propane mixture changes with concentra-
tion of propane.

$$\ln \text{ mean } \Delta T = 158.5^{\circ}\text{F.}$$

$$U = \frac{1}{\frac{1}{1,000} + \frac{1}{282}} = 220 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F.})$$

$$\text{Area} = \frac{858,100}{220 \times 158.5} = 16.0 \text{ sq.ft.}$$

$$\text{Cost} = 16.0 \times 15.00 \times 1.07 = \$256$$

E5:

Hot side: propane vapor from
flash drum D2

$$t_{\text{condensing}} = 152^{\circ}\text{F.}$$

Cold side: cooling water

$$t_{\text{in}} = 90^{\circ}\text{F.}$$

$$t_{\text{out}} = 105^{\circ}\text{F.}$$

$$\text{Duty} = 58,330 (106.6) = 6,210,000 \text{ B.t.u./hr.}$$

$$\begin{aligned} \text{Cooling-water consumption} &= \frac{6,210,000}{15 \times 8.33} \\ &= 49,700 \text{ gal./hr.} \end{aligned}$$

$$\ln \text{ mean } \Delta T = 54.5^{\circ}\text{F.}$$

$$U = 171.5 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F.})$$

$$\text{Area} = \frac{6,210,000}{54.5 \times 171.5} = 665 \text{ sq.ft.}$$

$$\text{Cost} = 665 \times 4.90 \times 1.18 = \$3,840$$

E6:

Hot side: propane vapor from
flash drum D3 4,624 lb./hr. at 252^oF.
flash drum D4 204.7 lb./hr. at 237^oF.
compressor C1 309.3 lb./hr. at 165^oF.

Cold side: cooling water

$$t_{\text{in}} = 90^{\circ}\text{F.}$$

$$t_{\text{out}} = 105^{\circ}\text{F.}$$

Average vapor temperature in

$$\begin{aligned} &= \frac{4,624 \times 252 + 204.7 \times 237 + 309.3 \times 665}{4,624 + 204.7 + 309.3} \\ &= 247^{\circ}\text{F.} \end{aligned}$$

Heat capacity assumed constant over range
involved.

Condensing temperature at 230 lb./sq.in.
abs. = 116^oF.

$$\text{Desuperheating duty} = 5,138 \times 0.5(247 - 116) \\ = 336,000 \text{ B.t.u./hr.}$$

$$\begin{aligned} \text{Condensing duty} &= 5,138(127.8) \\ &= 655,000 \text{ B.t.u./hr.} \end{aligned}$$

$$\begin{aligned} \text{Total duty} &= 336,000 + 655,000 \\ &= 991,000 \text{ B.t.u./hr.} \end{aligned}$$

$$\begin{aligned} \text{Cooling-water consumption} &= \frac{991,000}{(105 - 90)8.33} \\ &= 7,940 \text{ gal./hr.} \end{aligned}$$

Desuperheating:

$$\ln \text{ mean } \Delta T = \frac{247 - 105 - (116 - 90)}{\ln \frac{247 - 105}{116 - 90}} = 68^{\circ}\text{F.}$$

(Judging Committee comment: Adding
liquid propane to superheated vapor
until temperature of resulting vapor
is such that tube wall is at or below
condensing temperature makes it
possible to increase heat transfer
coefficient and thus lower apprecia-
bly surface area.)

$$U = \frac{1}{\frac{1}{15} + \frac{1}{300}} = 14.3 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F.})$$

$$\text{Area} = \frac{Q}{U \cdot \Delta T} = \frac{336,000}{14.3 \cdot 68} = 345 \text{ sq.ft.}$$

Condensing:

$$\ln \text{ mean } \Delta T = 17.5^{\circ}\text{F.}$$

$$U = \frac{1}{\frac{1}{400} + \frac{1}{300}} = 171.5 \text{ B.t.u./hr.}(sq.ft./^{\circ}\text{F.})$$

$$\text{Area} = \frac{655,000}{17.5 \times 171.5} = 218 \text{ sq.ft.}$$

$$\begin{aligned} \text{Total area} &= 345 + 218 = 563 \text{ sq.ft.} \\ \text{Cost} &= 563 \times 6.15 \times 1.07 = \$3,740 \end{aligned}$$

E7:

Hot side: color bodies from stripper S2

$$t_{\text{in}} = 237^{\circ}\text{F.}$$

$$t_{\text{out}} = 125^{\circ}\text{F.}$$

Cold side: cooling water

$$t_{\text{in}} = 90^{\circ}\text{F.}$$

$$t_{\text{out}} = 105^{\circ}\text{F.}$$

The storage temperature of color bodies
is set at 125^oF.

$$\text{Duty} = 430 \times 0.50(237 - 125) = 24,050 \text{ B.t.u./hr.}$$

$$\begin{aligned} \text{Cooling-water consumption} &= \frac{24,050}{15 \times 8.33} \\ &= 193 \text{ gal./hr.} \end{aligned}$$

$$\ln \text{ mean } \Delta T = 73^{\circ}\text{F.}$$

$$U = \frac{1}{\frac{1}{300} + \frac{1}{50}} = 42.9 \text{ B.t.u./hr.}(sq.ft./^{\circ}F.)$$

The h for color bodies is assumed to be the same for tallow.

$$\text{Area} = \frac{24,050}{42.9 \times 73} = 7.7 \text{ sq.ft.}$$

$$\text{Cost} = 7.7 \times 15.00 \times 0.94 = \$109$$

E8:

Hot side: 150 lb./sq.in. gauge steam
t = 366^oF.

Cold side: color bodies and propane from extraction tower

$$t_{in} = 165^{\circ}F.$$

$$t_{out} = 237^{\circ}F.$$

It is assumed that the boiling point of the propane solution of the color bodies changes with concentration. The temperature drop on throttling is neglected in the calculation of ΔT .

$$\text{Duty} = 430 \times 0.50(237-165) + 204.7(465-312) + 13.3 \times 0.5(237-165) = 44,500 \text{ B.t.u./hr.}$$

$$\text{Steam consumption} = \frac{44,500}{857} = 52 \text{ lb./hr.}$$

$$\ln \text{ mean } \Delta T = 165^{\circ}F.$$

$$U = \frac{1}{\frac{1}{1,000} + \frac{1}{277}} = 217 \text{ B.t.u./hr.}(sq.ft./^{\circ}F.)$$

$$\text{Area} = \frac{44,500}{217 \times 165} = 1.24 \text{ sq.ft.}$$

$$\text{Cost} = 1.3 \times 15.00 \times 1.46 = \$28$$

E9:

Hot side: tallow from stripper S1
t_{in} = 252^oF.

Cold side: raw tallow from storage

$$t_{in} = 125^{\circ}F.$$

$$t_{out} = 165^{\circ}F.$$

$$\text{Duty} = 8,600 \times 0.529(165-125) = 182,000 \text{ B.t.u./hr.}$$

$$t_{out} \text{ of hot stream} = 252 - \frac{182,000}{8,170 \times 0.53} = 210^{\circ}F.$$

$$\ln \text{ mean } \Delta T = 86^{\circ}F.$$

$$U = \frac{1}{\frac{1}{50} + \frac{1}{50}} = 25 \text{ B.t.u./hr.}(sq.ft./^{\circ}F.)$$

$$\text{Area} = \frac{182,000}{25 \times 86} = 85 \text{ sq.ft.}$$

$$\text{Cost} = 85 \times 15.00 \times 1.46 = \$1,820$$

E10:

Hot side: tallow from stripper S1 after passing through E9

$$t_{in} = 210^{\circ}F.$$

$$t_{out} = 125^{\circ}F.$$

Cold side: cooling water

$$t_{in} = 90^{\circ}F.$$

$$t_{out} = 105^{\circ}F.$$

$$\text{Duty} = 8,170 \times 0.53(210-125) = 368,000 \text{ B.t.u./hr.}$$

$$\text{Cooling-water consumption} = \frac{368,000}{15 \times 8.33}$$

$$= 2,940 \text{ gal./hr.}$$

$$\ln \text{ mean } \Delta T = \frac{210-105-(125-90)}{\ln \frac{210-105}{125-90}} = 63.6^{\circ}F.$$

$$U = \frac{1}{\frac{1}{50} + \frac{1}{300}} = 42.9 \text{ B.t.u./hr.}(sq.ft./^{\circ}F.)$$

$$\text{Area} = \frac{368,000}{42.9 \times 63.6} = 135 \text{ sq.ft.}$$

$$\text{Cost} = 135 \times 10.50 \times 0.94 = \$1,335$$

E11:

Hot side: exhaust steam, 15 lb./sq.in. gauge
t = 250^oF.

Cold side: propane from storage tank
t_{out} = 165^oF.

The propane storage temperature will be the average temperature of the streams flowing into the storage tank. Heat losses from the storage tank will be negligible because the throughput is large in relation to the volume of the tank.

Propane-storage temperature

$$= \frac{58,330 \times 175 + 5,138 \times 116 + 58,330 \times 152}{58,330 + 5,138 + 58,330}$$

$$= 161.4^{\circ}F.$$

Constant heat capacity assumed over range involved.

$$\text{Duty} = 121,800 \times 0.5(165-161.4) = 219,000 \text{ B.t.u./hr.}$$

Fifteen-pound exhaust steam will be used for heating.

Exhaust steam is 89.5% quality.

$$\text{Exhaust-steam consumption} = \frac{219,000}{945 \times 0.895}$$

$$= 259 \text{ lb./hr.}$$

This amount is available from P1.

$$\ln \text{ mean } \Delta T = \frac{250-161.4-(250-165)}{\ln \frac{250-161.4}{250-165}} = 86.8^{\circ}F.$$

$$U = \frac{1}{\frac{1}{1,000} + \frac{1}{120}} = 107 \text{ B.t.u.}/(\text{hr.})(\text{sq.ft.}/^{\circ}\text{F.})$$

$$\text{Area} = \frac{219,000}{107 \times 86.8^{\circ}\text{F.}} = 23.4 \text{ sq.ft.}$$

$$\text{Cost} = 23.4 \times 15.00 \times 1.46 = \$512$$

VIII. Compressor C1

At the rate of 309.3 lb./hr. propane vapor enters at 105°F. and 15 lb./sq.in.abs. and leaves at 165°F. and 230 lb./sq.in.abs. Compressor is three stage with intercoolers.

The calculations are made by the method described in reference 4 (Mollier diagram for propane used from reference 1).

Stage	Pressure, lb./sq.in.abs.	$(H_2 - H_1)_S$, B.t.u./lb.	Temperature after compression, °F.
1	15 to 38	22	155
2	36 to 93	22	160
3	91 to 230	22	165

$$\text{Total } (H_2 - H_1)_S = 66 \text{ B.t.u./lb.}$$

$$(H_2 - H_1) \text{ actual} = \frac{66}{0.75} = 88 \text{ B.t.u./lb.}$$

$$\begin{aligned} \text{Compressor rating} &= 88 \times \frac{309.3}{60} \times \frac{778}{33,000} \\ &= 10.7 \text{ hp.} \end{aligned}$$

$$\text{Capacity} = \frac{309.3}{60 \cdot 44} \times 359 = 42.0 \text{ std. cu.ft./min.}$$

Heat removed in intercoolers

$$\begin{aligned} &= 309.3 \left[0.4(105-100) + 2 \frac{22}{0.75} \right] \\ &= 18,750 \text{ B.t.u./hr.} \end{aligned}$$

$$\begin{aligned} \text{Cooling-water consumption} &= \frac{18,750}{1(105-90)8.33} \\ &= 150 \text{ gal./hr.} \end{aligned}$$

Steam drive:

$$\begin{aligned} \text{Power per pound of steam} &= 0.60(1,195.6 - 106.5)0.000393 \\ &= 0.0308 \text{ hp.-hr./lb.} \end{aligned}$$

$$\begin{aligned} \text{150-lb. steam consumption} &= \frac{10.7}{0.0308} \\ &= 347 \text{ lb./hr.} \end{aligned}$$

$$\text{Compressor cost} = 10.7 \times 100 = \$1,070$$

IX. Pumps

In the calculation of the pumps, friction and velocity heads are necessarily neglected. It

is assumed that the pump cost is based on horsepower delivered to liquid.

(Judging Committee comment: Horsepower should be figured on installed basis.)

P1 - raw tallow to extraction tower, 8,600 lb./hr.

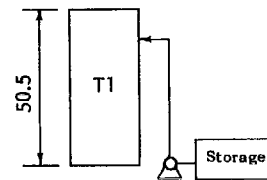
$$P_{in} = 14.7 \text{ lb./sq.in.abs.}$$

$$P_{out} = 450 \text{ lb./sq.in.abs.}$$

Power delivered to liquid

$$= q [x_2 - x_1 + v(P_2 - P_1)]$$

$$\begin{aligned} &= \frac{8,600}{3,600} \left[50.5 + \frac{(450-14.7)144}{[0.906-3 \times 10^{-4}(125-100)]} \right] \frac{1}{550} \\ &= 5.05 \text{ hp.} \end{aligned}$$



$$\text{Input to pump} = \frac{5.05}{0.60} = 8.41 \text{ hp.}$$

$$\begin{aligned} \text{150-lb. steam consumption} &= \frac{8.41}{0.0308} \\ &= 273 \text{ lb./hr.} \end{aligned}$$

$$\text{Capacity} = \frac{8,600}{60} \cdot \frac{1}{8.33 \times 0.90} = 19.1 \text{ gal./min.}$$

$$\text{Pump cost} = 5.05 \times 115 = \$580$$

P2 - propane to extraction tower, 121,800 lb./hr.

Difference in elevation is neglected.

$$P_{in} = 445 \text{ lb./sq.in.abs.}$$

$$P_{out} = 459 + 5 = 464 \text{ lb./sq.in.abs.}$$

Tower pressure drop = 5 lb./sq.in.

$$\text{Power to liquid} = q \times v(P_2 - P_1)$$

$$\begin{aligned} &= \frac{121,800}{3,600} 0.041(464-445) \frac{144}{550} \\ &= 5.09 \text{ hp.} \end{aligned}$$

$$\text{Input to pump} = \frac{5.09}{0.60} = 8.48 \text{ hp.}$$

$$\begin{aligned} \text{150-lb. steam consumption} &= \frac{8.48}{0.0308} \\ &= 275 \text{ lb./hr.} \end{aligned}$$

$$\text{Capacity} = \frac{121,800}{60 \times 3.2 \text{ lb./gal.}} = 650 \text{ gal./min.}$$

$$\text{Cost} = 5.09 \times 115 = \$585$$

P3 - propane from holding tank H2, 58,330 lb./hr.

Difference in elevation is assumed negligible.

$$P_{in} = 349 \text{ lb./sq.in.abs.}$$

$$P_{out} = 449 \text{ lb./sq.in.abs.}$$

Power to liquid

$$= \frac{58,330}{3,600} \times 0.0385(449-349)144 \times \frac{1}{550}$$

$$= 16.2 \text{ hp.}$$

$$\text{Input to pump} = \frac{16.2}{0.60} = 27.0 \text{ hp.}$$

$$150\text{-lb. steam consumption} = \frac{27.0}{0.0308}$$

$$= 876 \text{ lb./hr.}$$

$$\text{Capacity} = \frac{58,330}{60} \times \frac{1}{3.4 \text{ lb./gal.}} = 286 \text{ gal./min.}$$

$$\text{Cost} = 16.2 \times 96 = \$1,555$$

P4 - propane from holding tank H1 to storage, 5,138 lb./hr.

Differences in elevation are assumed negligible.

$$P_{in} = 230 \text{ lb./sq.in.abs.}$$

$$P_{out} = 445 \text{ lb./sq.in.abs.}$$

Power to liquid

$$= \frac{5,138}{3,600} \times 0.035(445-230) \frac{144}{550} = 2.81 \text{ hp.}$$

$$\text{Input to pump} = \frac{2.81}{0.60} = 4.69 \text{ hp.}$$

$$150\text{-lb. steam consumption} = \frac{4.69}{0.0308}$$

$$= 152 \text{ lb./hr.}$$

$$\text{Capacity} = \frac{5,138}{60} \times \frac{1}{3.5} = 24.4 \text{ gal./min.}$$

$$\text{Cost} = 2.81 \times 190 = \$534$$

X. Holding Tanks - designed for 5 min. holding time.

$$H1 - \text{flow} = 5,138 \times \frac{1}{12} = 428 \text{ lb./5 min.}$$

$$\text{Sp. vol. propane} = 0.035 \text{ cu.ft./lb.}$$

$$\text{Volume of holding tank} = 428 \times 0.035 = 18.3 \text{ cu.ft.}$$

Diameter arbitrarily fixed at 2 ft.

$$\text{Height} = \frac{18.3}{\left(\frac{2}{2}\right)^2} = 5.83 \text{ ft.}$$

round off to 6.0 ft.

$$\text{Cost} = 6 \times 28 \times 2.20 = \$370$$

$$\text{Operating pressure} = 230 \text{ lb./sq.in.abs.}$$

$$H2 - \text{flow} = 58,330 \times \frac{1}{12} = 4,855 \text{ lb./5 min.}$$

$$\text{Sp. vol.} = 0.0385 \text{ cu.ft./lb.}$$

$$\text{Volume of tank} = 4,855 \times 0.0385 = 187 \text{ cu.ft.}$$

Diameter arbitrarily fixed at 5 ft.

$$\text{Height} = \frac{187}{\pi \left(\frac{5}{2}\right)^2} = 9.5 \text{ ft.}$$

$$\text{Operating pressure} = 349 \text{ lb./sq.in.abs.}$$

$$\text{Cost} = 9.5 \times 74 \times 2.70 = \$1,900$$

XI. Propane Inventory

$$\text{Volume of extraction tower} = 50.5 \times \pi \left(\frac{8.5}{2}\right)^2 = 2,870 \text{ sq.ft.}$$

Propane in extraction tower

$$= \frac{121,800}{121,800 + 8,600} 2,870 \times 24.4 = 65,400 \text{ lb.}$$

The flash drums and holding tanks are assumed to run normally at one half their holding capacity.

$$\text{Propane in D1} = 63,250 \times 1/12 \times 1/2 = 2,640 \text{ lb.}$$

$$D2 = 4,920 \times 1/12 \times 1/2 = 205 \text{ lb.}$$

$$D3 = 296 \times 1/12 \times 1/2 = 12.5 \text{ lb.}$$

$$D4 = 13.3 \times 1/12 \times 1/2 = 0.5 \text{ lb.}$$

$$H1 = 1/2 \times 428 = 214 \text{ lb.}$$

$$H2 = 1/2 \times 4,855 = 2,428 \text{ lb.}$$

$$\text{Storage tank} = 25\pi \left(\frac{9}{2}\right)^2 \times \frac{1}{2} \times 24.4 = 19,400 \text{ lb.}$$

The amount of propane in the piping and heat exchangers cannot be calculated and is therefore neglected.

$$\text{Total propane in system} = 88,300 \text{ lb.}$$

$$\text{Density of liquid at } 60^\circ\text{F.} = 4.24 \text{ lb./gal.}$$

$$\text{Original propane cost} = \frac{88,300}{4.24} \times 0.06 = \$1,250$$

XII. Utilities

150-lb. steam consumption

$$\text{Steam strippers } S1 = 300 \text{ lb./hr.}$$

$$S2 = 100 \text{ lb./hr.}$$

$$\text{Compressor } C1 = 347 \text{ lb./hr.}$$

$$\text{Heat exchangers } E1 = 7,370 \text{ lb./hr.}$$

$$E4 = 1,000 \text{ lb./hr.}$$

$$E8 = 52 \text{ lb./hr.}$$

$$\text{Pumps } P1 = 273 \text{ lb./hr.}$$

$$P2 = 275 \text{ lb./hr.}$$

$$P3 = 876 \text{ lb./hr.}$$

$$P4 = 152 \text{ lb./hr.}$$

$$\text{Instrument compressor} = 25 \text{ lb./hr.}$$

$$\text{Total 150-lb. steam}$$

$$\text{consumption} = 10,440 \text{ lb./hr.}$$

$$\text{Cost per year}$$

$$= 365 \times 0.85 \times 24 \times 10,440 \times 0.00045$$

$$= \$35,000$$

$$\text{Additional cost for stripping steam}$$

$$= 365 \times 0.85 \times 24 \times 400 \times 0.00010 = \$298$$

15-lb. steam consumption

$$\text{Heat Exchanger } E11 = 259 \text{ lb./hr.}$$

$$\text{Building and pump house heating}$$

$$= 200 \text{ lb./hr.}$$

$$\text{Steam for storage facilities} = 400 \text{ lb./hr.}$$

$$(\text{cost of this item is to be neglected})$$

Pump and compressor exhaust steam is to be used for these purposes.

Consumption of 15-lb. steam from boiler
 = 0 lb./hr.
 Total cost of steam per year = \$35,300

Cooling-water consumption:
 Heat exchangers E3 = 1,710 gal./hr.
 E5 = 49,700 gal./hr.
 E6 = 7,940 gal./hr.
 E7 = 193 gal./hr.
 E10 = 2,940 gal./hr.

Compressor C1 inter-coolers = 150 gal./hr.
 Total cooling-water consumption = 62,633 gal./hr.
 Annual cooling-water cost
 = $365 \times 0.85 \times 24 \times 62,633 \times 0.00003$
 = \$14,000

City-water consumption in jet condenser
 = 1,820 gal./hr.
 Annual city-water cost
 = $365 \times 0.85 \times 24 \times 1,820 \times 0.00008$
 = \$1,083

Total water cost per year = \$15,100

Jet condenser J1 = \$ 222
 Heat exchangers E1 = 2,485
 E2 = 11,700
 E3 = 350
 E4 = 256
 E5 = 3,840
 E6 = 3,740
 E7 = 109
 E8 = 28
 E9 = 1,820
 E10 = 1,335
 E11 = 512

Compressor C1 = 1,070
 Pumps P1 = 580
 P2 = 585
 P3 = 1,555
 P4 = 534

Holding tanks H1 = 370
 H2 = 1,900

Propane = 1,250

Total basic-process-equipment cost
 = \$78,650
 Auxiliary-equipment-material cost
 = $78,650 \times 0.305 = \$24,000$
 Freight, erection labor, taxes, etc.
 = $(78,650 + 24,000) \times 0.21 = \$21,550$
 Electrical Power
 Power for lighting, etc. = 35 kw.
 Cost per year
 = $365 \times 0.85 \times 24 \times 35 \times 0.01 = \$2,600$
 Total annual utilities cost
 = $35,300 + 15,100 + 2,600 = \$53,000$
 Propane
 Propane loss = 121,800 0.0003
 = 36.54 lb./hr.
 Cost to replace propane lost
 = $\frac{36.54}{4.24} \times 0.06 = \$0.517/\text{hr.}$
 Annual propane cost
 = $365 \times 0.85 \times 24 \times 0.517 = \$3,850$

XIII. Total Costs

Basic-process-equipment-material cost
 Extraction tower T1 = \$35,384
 Flash drums D1 = 5,623
 D2 = 1,580
 D3 = 697
 D4 = 186
 Steam strippers S1 = 397.50
 S2 = 397.50

Total direct cost
 = $78,650 + 24,000 + 21,550 = \$124,200$
 Total indirect cost
 = $124,200 \times 0.12 = \$14,910$
 Constructed cost
 = $124,200 + 14,910 = \$139,110$
 Total annual fixed charges
 = $139,110 \times 0.22 = \$30,600$

XIV. Cost Summary

Total costs per year
 Fixed charges = \$30,600
 Utilities = 53,000
 Propane = 3,850
 Labor = 95,000
 Maintenance = 35,000
 Total annual cost \$217,450

Annual revenues
 Color bodies
 = $365 \times 0.85 \times 24 \times 430 \times 0.01 = \$32,000$
 Net annual cost \$185,450

Tallow produced per year
 = $365 \times 0.85 \times 24 \times 8,170 = 60,800,000 \text{ lb.}$
 Cost per lb. of tallow produced
 = $\frac{185,450}{60,800,000} = \0.00305

Annual cost of caustic process
 = $60,800,000 \times 0.00499 = \$303,500$
 Annual saving of extraction over caustic
 process = $303,500 - 185,450 = \$118,050$
 % saving = $\frac{0.499 - 0.305}{0.499} \times 100 = 39\%$

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